

โครงการ

การออกแบบระบบควบคุมโรบัสแบบดีสครีตของตัวชดเชยสเตติกช์ชิง โครนัสแบบอนุกรมเพื่อการควบคุมเสถียรภาพของการแกว่งของ ความถี่ในระบบไฟฟ้ากำลังที่เชื่อมโยง

Robust Discrete Control Design of Static Synchronous Series Compensator (SSSC) for Stabilization of Frequency Oscillations in an Interconnected Power System

โดย

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มีถุนายน 2546



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รายงานวิจัยฉบับสมบูรณ์

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คณะผู้วิจัย

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เพื่อการควบคุมเสถียรภาพของการแกว่งของความถี่ในระบบใฟฟ้ากำลังที่เขื่อมใยง

Robust Discrete Control Design of Static Synchronous Series Compensator (SSSC) for Stabilization of Frequency Oscillations in an Interconnected Power System

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Abstract

As an AC interconnected power system is subjected to a large load with rapid change, system frequency may be severely disturbed and becomes oscillatory. To stabilize the frequency oscillations, the dynamic power flow control of the Static Synchronous Series Compensator (SSSC) located in series with the tic line between interconnected power systems, is employed. By regarding the system interconnection as the control channel, the power flow control by an SSSC via the interconnection creates a sophisticated method of frequency stabilization. To implement this concept, the robust design method of the lead/lag controller equipped with the SSSC is proposed. In the design process, not only the attenuation performance of system disturbances, but also the robust stability against system uncertainties are taken into consideration. The optimal parameters of the lead/lag controller are obtained by using the tabu search algorithm (TSA), so that both performance and robustness are satisfied. The frequency stabilizer developed by the proposed design can be implemented in both continuous and discrete time domains. Simulation study exhibits the significant damping effects of designed frequency stabilizer on frequency oscillations under different load disturbances and negative damping.

บทดัดย่อ

ในปัจจุบัน ความต้องการทางด้านพลังงานใฟฟ้าและการแข่งขันกันระหว่างผู้ประกอบการใฟฟ้าอิสระมีแนวโน้มที่สงขึ้น อย่างมากทั่วโลกในสภาพการเช่นนี้ การควบคุมในระบบใพ่ฟ้ากำลังเช่น การควบคุมเสถียรภาพทางความถื่นละแรงดันเป็นต้น นั้นจะเปลี่ยนใปในรูปลักษณะเป็นการให้บริการที่เรียกว่า Ancillary Services โดยเฉพาะอย่างยิ่งในระบบใฟฟ้ากำลังที่มีการ ี่ดีตตั้งภาระ (load) ทางไฟฟ้าที่มีความต้องการกำลังไฟฟ้าอย่างบากและยังมีการเปลี่ยนแปลงภาระอย่างรวดเร็ว อย่างเช่น รถไฟฟ้าความเร็วสูง, โรงงานอุดสาทกรรมหนักขนาดใหญ่ เป็นต้น นั้นก่อให้เกิดปัญหาการแกว่งของความถือย่างรุนแรง ซึ่ง แม้แต่ระบบควบคุมเสถียรภาพทางความถี่ที่ใช้ในปัจจุบัน คือ ระบบโกเวอร์เนอร์นั้นก็ยังใม่สามารถลดความรูนแรงของการแกว่ง ของความถี่ได้ การแกร่งของความถื่อย่างรุนแรงนี้จะส่งผลเสียต่อเสถียรภาพโดยรวมของระบบไฟฟ้ากำลังโดยตรงซึ่งจะทำให้ ระบบนั้นไม่สามารถจ่ายกำลังใพ่ฟ้าใต้ นอกจากนี้ แนวในมการเพิ่มขึ้นอย่างสูงของผู้ผลิตใพ่ฟ้าอีสระ ซึ่งความสามารถในการ ควบคุมความถี่ไม่เพียงพอกีมีผลเสียอย่างมากต่อเสถียรภาพทางความถึ่ของระบบ ภายใต้สถานการณ์ดังกล่าวข้างดันวิธีการ ให้บริการการควบคุมเสถียวภาพทางความถึ้แบบใหม่ ซึ่งมีประสิทธิภาพสูงกว่าระบบควบคุมที่ใช้ในปัจจุบันจึงมีความต้องการ งานวิจัยนี้นำเสนอการนำอุปกรณ์ทางอิเลดทรอนิกส์กำลังซึ่งมีความเร็วในการควบคุมสูงที่เรียกว่า Synchronous Series Compensator, (SSSC) ซึ่งมีความสามารถในการควบคุมการใหลของกำลังใฟฟ้าในสายเชื่อมโยง และสามารถนำมาประยุกดีใช้เพื่อควบคุมเสถียวภาพทางการแกว่งของความถึงองระบบใฟฟ้ากำลังใต้อย่าง รวดเร็ว โดยงานวิจัยนี้มุ่งเน้นใปในด้านการออกแบบระบบควบคุมโรบัสแบบต่อเนื่องและดีสหรืดของ SSSC นอกจากเพื่อให้มี ประสิทธิภาพในการอดความรุนแรงของการแกว่งของการแกว่งของความก็และยังเพิ่มคุณสมบัติ Robustness ของตัวควบคุม ต่อกวามไม่แน่นอนต่างๆซึ่งจะเกิดขึ้นในระบบ (system uncertainties) ให้เหนือกว่าตัวควบคุมที่ไม่มีคุณสมบัตินี้ นอกจากนี้ โครงสร้างของระบบควบคุมที่ใช้ในงานวิจัยนี้มีอักษณะเป็นตัวขณาขอแบบ second-order lead/lag จึงเป็นการง่ายในการ นำมาใช้งานจริง ในงานวิจัยนี้ ได้นำ Tabu Scarch Algorithm มาใช้สำหรับการหาคำพารามิเตอร์ที่เหมาะสมที่สุดของตัว ควบคุม ผลการจำลองศึกษาแสดงให้เห็นถึงประสิทธิภาพและคุณสมบัติ Robustness ของตัวควบคุมที่นำเสนอ

Keywords: Flexible AC Transmission Systems (FACTS), Static synchronous series compensator (SSSC), robust control, system uncertainties, tabu search algorithm, overlapping decompositions, power system stabilization, frequency oscillations, ancillary services.

หน้าสรุปโครงการ (Executive Summary)

1. ปัญหาที่ทำการวิจัย และความสำคัญของปัญหา

ในปัจจุบันความต้องการทางด้านพลังงานไฟฟ้าและการแข่งขันกันระหว่างผู้ประกอบการไฟฟ้าอิสระมีแนวโน้มที่สูงขึ้น อย่างมากทั่วโลก ในสภาพการเช่นนี้ การดวบคุมในระบบไฟฟ้ากำลังเช่น การดวบคุมเสถียรภาพทางความถี่และแรงดันเป็นต้น นั้นจะเปลี่ยนไปในรูปลักษณะเป็นการให้บริการที่เรียกว่า Ancillary Services โดยเฉพาะอย่างยิ่งในระบบไฟฟ้ากำลังที่มีการ ดิดตั้งภาระ (load) ทางไฟฟ้าที่มีความต้องการกำลังไฟฟ้าอย่างมากและยังมีการเปลี่ยนแปลงภาระอย่างรวดเร็ว อย่างเช่น รถไฟฟ้าความเร็วสูง, โรงงานอุตสาหกรรมหนักขนาดใหญ่ เป็นดัน นั้นก่อให้เกิดปัญหาการแกว่งของความถี่อย่างรุนแรง ซึ่ง แม้แต่ระบบควบคุมเสถียรภาพทางความถี่ที่ใช้ในปัจจุบันคือระบบโกเวอร์เนอร์นั้นก็ยังไม่สามารถลดความรุนแรงของการแกว่ง ของความถี่ได้ การแกว่งของความถี่อย่างรุนแรงนี้จะส่งผลเสียต่อเสถียรภาพโดยรวมของระบบไฟฟ้ากำลังโดยตรงซึ่งจะทำให้ ระบบนั้นไม่สามารถจ่ายกำลังไฟฟ้าได้ นอกจากนี้ แนวโน้มการเพิ่มขึ้นอย่างสูงของผู้ผลิตไฟฟ้าอิสระ (Independent Power Producers, IPPs) ซึ่งความสามารถในการควบคุมความถี่ไม่เพียงพอก็มีผลเสียอย่างมากต่อเสถียรภาพทางความถี่ของระบบ ภายใต้สถานการณ์ดังกล่าวข้างดันวิธีการให้บริการการควบคุมเสถียรภาพทางความถี่แบบใหม่ซึ่งมีประสิทธิภาพสูงกว่าระบบ ควบคุมที่ใช้ในปัจจุบันจึงมีความด้องการเป็นอย่างสูง

งานวิจัยนี้นำเสนอการนำอุปกรณ์ทางอิเลคทรอนิกส์กำลังซึ่งมีความเร็วในการควบคุมสูงที่เรียกว่า Static Synchronous Series Compensator, SSSC ซึ่งมีความสามารถในการควบคุมการใหลของกำลังให้ฟ้าในสายเชื่อมโยง แบบใดนามิกส์และสามารถนำมาประยุกด์ใช้เพื่อควบคุมเสถียรภาพทางการแกว่งของความถี่ของระบบให้ฟ้ากำลังได้อย่าง รวดเร็ว โดยงานวิจัยนี้มุ่งเน้นใปในด้านการออกแบบระบบควบคุมโรบัสแบบแบบต่อเนื่องและดีสครีตของ SSSC นอกจาก เพื่อให้มีประสิทชิภาพในการลดความรุนแรงของการแกว่งของการแกว่งของความถี่และยังเพิ่มคุณสมบัติ Robustness ของตัว ควบคุมต่อความไม่แน่นอนต่างๆซึ่งจะเกิดขึ้นในระบบ (System Uncertainties) ให้:::นือกว่าตัวควบคุมที่ไม่มีคุณสมบัติที่ไม่ มีคุณสมบัตินี้

2. วัตถุประสงค์

เพื่อศึกษาถึงปัญหาในการออกแบบตัวควบคุมโรบัสแบบค่อเนื่องและดีสครีตของ SSSC พร้อมวิธีแก้ไขปัญหา เพื่อใช้ ในการควบคุมเสถียรภาพของการแกว่งของความถี้ในระบบไฟฟ้ากำลังที่เชื่อมโยงกันในลักษณะต่าง ๆ เช่น Loop, Radial เป็นต้น

- 2.1. เพื่อศึกษาวิธีการออกแบบดัวควบคุมโรบัสแบบต่อเนื่องและดีสครีตของ SSSC เพื่อให้ได้ตัวควบคุมโรบัสที่มีประ-สิทธิภาพสูงและมีโครงสร้างที่ไม่ซับซ้อนเพื่อที่จะสามารถนำมาประยุกดีในการใช้งานจริงได้
- 2.2. เพื่อศึกษาถึงแนวทางการทำงานร่วมกันระหว่างตัวควบคุมโรบัสของ SSSC ที่นำเสนอและระบบควบคุมเสถียรภาพ ทางความถี่ที่ใช้กันอยู่ในปัจจุบันคือ ระบบโกเวอร์เนอร์
- 2.3. เพื่อศึกษาเปรียบเทียบความแตกต่างระหว่างตัวควบคุมโรบัสที่ถูกออกแบบโดยวิธีที่นำเสนอ และตัวควบคุมที่ไม่มี คุณสมบัติ Robustness

3. ระเบียบวิธีวิจัย

- 3.1 ตั้งปัญหาการออกแบบตัวควบคุมโรบัสแบบต่อเนื่องของ SSSC ในระบบไฟฟ้ากำลังแบบเชื่อมโยงซึ่งเกิดการแกว่ง ของความถื่อย่างรุนแรงจากสาเหตุต่าง ๆ เช่น การเปลี่ยนแปลงภาระไฟขนาดใหญ่อย่างรวดเร็ว เป็นตัน
- 3.2 สร้างแบบจำลองคณิตศาสตร์ของตัวควบคุมโรบัสแบบต่อเนื่องของ SSSC ในระบบไฟฟ้ากำลังแบบเชื่อมโยงที่กำลัง พิจารณา โดยรวมไปถึงแบบจำลองทางคณิตศาสตร์ซึ่งแทนความไม่แน่นอนในระบบที่เกิดขึ้น
- 3.3 ศึกษาแนวความคิดในการออกแบบตัวควบคุมของ SSSC รวมไปถึงการนำทฤษฎีระบบควบคุมโรบัฒนบบต่อเนื่อง มาประยุกต์ใช้ในการออกแบบตัวควบคุมของ SSSC
- 3.4 ออกแบบตัวควบคุมโรบัสแบบต่อเนื่องของ SSSC
- 3.5 เปรียบเทียบตัวควบคุมโรบัสแบบต่อเนื่องที่ถูกออกแบบมากับตัวควบคุมแบบต่อเนื่องที่ไม่มีคุณสมบัติ Robustness หลังจากนั้นทำการรวบรวมผลการทดสอบตัวควบคุมนำมาเขียนเป็นบทความวิชาการเพื่อลงในวารสาร Electric Power System Research และใต้รับการขอมรับในการสีพิมพ์เรียบร้อยแล้ว
- 3.8 ศึกษาทฤษฎีการออกแบบตัวควบคุมแบบดีสครีต พลังจากนั้นทำการออกแบบตัวควบคุมแบบดีสครีต โดยการพัฒนา

- มาจากการออกแบบตัวควบคุมแบบต่อเนื่อง
- 3.7 นำตัวควบคุมแบบดีสครีตที่ทำการออกแบบมาแด้วมาทศสอบประสิทธิภาพ เปรียบเทียบกับตัวควบคุมแบบดีสครีตที่ ไม่มีคุณสมบัติ Robustness ในด้านค่างๆ อาทิ เช่น ความทนทานต่อความไม่แน่นอนในระบบ, ความสามารถใน การลดความรุนแรงในการแกว่งของความถี่ในระบบไฟฟ้ากำลัง เป็นต้น แล้วนำผลการวิจัยที่ได้จากขั้นตอนดังกล่าว นำมาเขียนเป็นบทความวิชาการเพื่อลงในวารสาร ASEAN Journal on Science & Technology for Development ซึ่งได้ส่งบทความไปแล้ว กำลังรอผลการ Review อยู่

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Contents

1.	Introduction	1
	1.1 Motivation of The Research	i
	1.2 Current Technology and Problems of Stabilization of Frequency	
	Oscillations in Interconnected Power System	2
	1.3 Purpose of This Report	
2.	Stabilization of Frequency Oscillations by Static Synchronous Series Compensate 2.1 Introduction	or 4
	2.2 Concept and Practical Motivations of Proposed Control Strategy	5
	2.3 Mathematical Model of SSSC	
	2.4 Structure of Frequency Stabilizer	
3.	Design of Robust Frequency Stabilizer of SSSC in Continuous Time Domain 3.1 Purpose of This Chapter	8
	3.2 Problem Statement	
	3.3 Design Methodology	
	3.3.1 Coordinated Control of SSSC and Governor	
	3.3.2 Linearized Power System Model	
	3.3.3 Model Reduction by Overlapping Decompositions	
	3.3.4 Determination of Objective Function	
	3.3.5 TSA for Parameter Determination	
	3.4 Simulation Results and Evaluation	
	3.5 Conclusions	
4.	Design of Robust Frequency Stabilizer of SSSC in Discrete Time Domain 4.1 Purpose of This Chapter	31
	4.2 Problem Statement	
	4.3 Design Methodology	
	4.3.1 Coordinated Control of SSSC and Governor	
	4.3.2 Linearized Power System Model	32
	4.3.3 Model Reduction by Overlapping Decompositions	
	4.3.4 Structure of Discrete-based Frequency Stabilizer	36
	4.3.5 Determination of Objective Function	37
	4.3.6 TSA for Parameter Determination	
	4.4 Experimental Results	
	4.5 Conclusions	
5.		
6. 7.	References List of Publications	57 60

Chapter 1

Introduction

1.1 Motivation of The Research

Nowadays, electric power systems all over the world are undergoing drastic restructuring. The trend is toward deregulating an industry that has traditionally been a regulated monopoly to allow for economic competition. The restructuring of the conventional vertically integrated utilities into separated generation, transmission and distribution companies has been accomplished first in United Kingdom and almost everywhere up to now. This not only prompts a heavy competition among utilities but also increase the diversity of resource options for customers [1, 2].

The major change that has happened in the deregulated power system is the proliferation of the non-utility generation such as Independent Power Producers (IPPs) that play an important role in participating in power transactions. In the case that many IPPs which have not sufficient abilities of frequency and voltage controls, have been concentrated in any power system, it is very important to consider that how the control services of system frequency and voltage should be implemented.

Furthermore, it is reasonably expected that the power system in the future will be more and more interconnected with significant inter-area power flows along tie lines. Interconnected tie lines have the benefits of providing inter-area support for abnormal conditions as well as transmission paths for contractual energy exchanges between areas. Under this situation, there is possibility of the occurrence of unbalanced area power demands which may cause instability problems of frequency and inter-area power oscillations. To overcome this problem, a dynamic control of tie line power flow with the aim of not only alleviating the heavy tie line power flows, but also stabilizing the system frequency oscillations against the sudden changes in load demands, is significantly required.

In a deregulated power industry, any power system controls such as frequency and voltage controls will be served as ancillary services [3-6]. Especially, stabilization service of frequency oscillations in an interconnected system becomes challenging when implemented in the future competitive environment [7,8]. A new stabilization service which emphasizes not only on efficiency, reliability and economics but also on advanced and improved control for satisfying the requirement of power system operation, is highly expected.

1.2 Current Technology and Problems of Stabilization of Frequency Oscillations in Interconnected Power System

At present, the control of system frequency is operated by the Automatic Generation Controls (AGC) strategy of governor system developed nearly half a century ago [9-10]. The primary objectives of AGC in multi-area interconnected system are to regulate frequency to the specified nominal value and to maintain the interchange power between interconnected areas at the scheduled values by adjusting the output of selected generators. This function is commonly referred to as Load-Frequency Control (LFC) [11-13]. Current LFC strategies are based on three popular methods, i.e., Tie Lie Bias Control (TBC), Flat Frequency Control (FFC) and Flat Tie Line Control (FTC). For example, the purpose of TBC scheme is to alleviate not only the system frequency deviation but also the tie line power deviation due to changes in load demand by adjusting and coordinating the generation power between interconnected areas.

The AGC strategy is widely accepted and is well entrenched in practice in many countries. Nevertheless, its performance has never been viewed as fully satisfactory. There are significant reasons which explain about its deficiencies and why AGC performance might be less than ideal as follows [9]. AGC strategy is incapable of absorbing an abrupt large power mismatch between the load demand and the generation power due to its slow response characteristics. Even in the case of the operation by governor free, it is unable to play any role in suppressing the magnitude of the transient frequency which occurs within seconds after a rapid change in generation or load in the system. In addition, the capability of AGC is also deteriorated by some inevitable system nonlinearities such as the influences of deadbands in mechanical linkage and transport delays.

In addition, in the view point of dynamics between interconnected system, there is a possibility that the low frequency oscillations of inter-area power flows occur due to the heavy power transfer in tie line and the unbalance of area power demands [14-15]. This may cause a serious instability problem of frequency oscillations in interconnected systems. AGC scheme is neither able nor can be expected to play any role in damping eletromechanical dynamic including inter-area oscillations [9]. Present AGC scheme may no longer be able to solve these problems. A new stabilization strategy of frequency oscillations in interconnected power system to provide not only a dynamic control of interconnected tie line power flow but also a fast stabilization of the transient frequency deviation is significantly required.

1.3 Purpose of This Report

Recently, the concept of utilizing power electronics devices for power system control has been purposed as FACTS (Flexible AC Transmission Systems) [16]. The FACTS concept [17-19] is to increase the capability, controllability and flexibility of power transfer of existing AC transmission facilities while maintaining the reliability of the power delivery. FACTS devices play an active role in determining real and reactive power flow by dynamic control of the power transfer parameters, i.e., transmission voltage, line impedance and phase angle. In other words, the AC transmission system is changed from a passive element to an active power delivery element. Among of them, a Static Synchronous Series Compensator [20-21] (SSSC) has been highly expected as an effective apparatus with an ability of fast power flow control. By virtue of this characteristic, SSSC is utilized for dynamically adjusting the network configuration to enhance steady state performance as well as transient stability in power systems.

In the AGC scheme, the interconnections among some areas have been considered as the channels of disturbances. Therefore, it has been important to suppress the frequency fluctuations by the coordination of governor systems among interconnected areas. On the contrary, the dynamic power flow control by the SSSC located in series with the tie line between two areas interconnected system, has the possibility to stabilize the system frequency through the interconnections. In this thesis, a new control method using an SSSC to provide a dynamic control of tie-line power flow for stabilization of frequency oscillations in AC interconnected power system is proposed. The active control of tie line power by SSSC is not only effective to stabilize the dynamic oscillatory behavior of the inter-are mode, but also capable of suppressing the transient frequency deviations. According to the proposed control, the power system which has a large frequency control capability, is able to offer the stabilization service of frequency oscillations to other interconnected areas which have not sufficient capabilities. The proposed frequency stabilization by SSSC can be expected to be an advanced method of interconnection of AC power systems competing against the interconnection by High Voltage Direct Current (HVDC) transmission system or Back To Back (BTB) system. Moreover, the proposed control is expected as a new ancillary service for stabilization of frequency oscillations in any interconnected systems which have insufficient control capabilities, instead of the installation of large power generation units for frequency control.

Chapter 2

Stabilization of Frequency Oscillations by Static Synchronous Series Compensator

2.1 Introduction

Flexible AC Transmission System (FACTS) [16-19] is a sophisticated technology sponsored by Electric Power Research Institute (EPRI) to help the utility industry in order to fully utilize available AC transmission system. A major thrust of FACTS technology is based on the utilization of reliable high-speed power electronics that provide dynamic control of the power transfer actively [22]. Among of FACTS devices, a phase shifter with a solid-state voltage source inverter using gate turn-off (GTO) thyristors [20] such as Static Synchronous Series Compensator (SSSC) [21], Unified Power Flow Controller (UPFC) [23] etc., has several attractive characteristics for power system applications. Especially, it is capable of emulating an inductive or a capacitive reactance, therefore, influences the power flow in the transmission directly. Moreover, even if the tie line power flow is very small, in other words the tie line current is near zero, a phase shifter still has effect on power flow control. This exhibits the superior characteristics of phase shifter beyond the series capacitance or reactance. The sophisticated characteristics of solid-state phase shifter motivate many power system applications, e.g., enhancement of transient stability [24], improvement of system power dynamics [25], power system stabilization [26], loop power flow control [27], damping of subsynchronous resonance [28] etc. Several studies about modeling and experiment with the aim of realizing the solid-state phase shifter in practice have been also reported [29-30]. Moreover, some types of solid-state phase shifter have been already operated in real power system [31].

In this report, a new control strategy using a Static Synchronous Series Compensator (SSSC) to provide an active control facility for stabilization of frequency oscillations in an interconnected power system is proposed. An SSSC located in series with the tie line between two areas interconnected system is capable of controlling the tie line power flow dynamically. Moreover, it has the possibility to stabilize the system frequency oscillations by fast tie line power modulations through the interconnections.

This chapter focuses on the concept and practical motivation of the proposed stabilization of frequency oscillations in an interconnected power system by SSSC. In addition, the derivation of mathematical model of SSSC is curried out.

2.2 Concept and Practical Motivations of Proposed Control Strategy

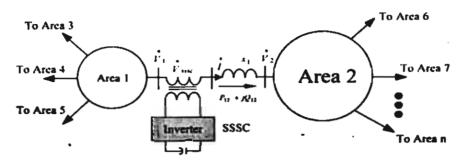


Figure.2.1: An SSSC in a multi-area interconnected power system

Figure 2.1 shows a general multi-area interconnected power system. Both areas have several interconnections other than area 1 or 2. A phase shifter is located in series with the tie line between areas 1 and 2. It is assumed that frequency control of area 1 is beyond its capability while the area 2 has the capability enough to spare for stabilization service of frequency oscillations in an area 1. Accordingly, the tie line power modulation by a phase shifter which is supported by the capability of area 2, is utilized to stabilize the frequency oscillations of area 1.

The practical motivations for application of the proposed phase shifter to stabilization of frequency oscillations are as follows,

- 1. In a competitive deregulated system, non-utility generations such as many Independent Power Producers (IPPs) which have not sufficient abilities of frequency control, tend to increase significantly. When they have been concentrated in an area, it is important to pay attention to how this area frequency should be controlled. Under this circumstance, the proposed phase shifter can utilize the frequency control capabilities of other interconnected areas to stabilize the frequency oscillation of this area. The proposed control strategy will be a new option of stabilization service of frequency oscillation for any power system which has not sufficient frequency control capabilities, instead of the installation of large power generation units for frequency control.
- Nowadays, various kinds of apparatus with large capacity and consumptions increase significantly; for typical examples, a magnetic levitation transportation, large scale accelerator, testing plant for nuclear fusion and so on. In addition, even in the case that an ordinary scale factory like a steel manufacture is installed in a small power system, it may cause a serious frequency fluctuation problem. Under this situation, the conventional governor systems, even in the case of the operation of governor free, may no longer be able to absorb the frequency oscillations. In order to compensate these

- sudden load perturbations and stabilize system frequency oscillations quickly, the proposed phase shifter is expected as one of the most effective countermeasure.
- 3. Due to the benefits of scale merit, the construction of power generation unit with large capabilities tends to increase considerably. This may cause a transient depressed frequency when the losses of generation units suddenly occur. To tackle this situation, the proposed control is very suitable option. Phase shifter is capable of borrowing the control capabilities of other interconnected areas in order to alleviate the transient frequency deviations rapidly.

2.3 Mathematical Model of SSSC

In this study, the mathematical model of the SSSC for stabilization of frequency oscillations is derived from the characteristic of power flow control by SSSC [20-21]. By adjusting the output voltage of SSSC (\bar{V}_{SSSC}), the tie-line power flow ($P_{12} + jQ_{12}$), can be directly controlled as shown in Fig.2.1. Since the SSSC fundamentally controls only the reactive power, then the phasor \bar{V}_{SSSC} is perpendicular to the phasor of line current \bar{I} , which can be expressed as

$$\bar{V}_{SSSC} = jV_{SSSC}.\bar{I}/I \tag{2.1}$$

where V_{SSSC} and I are the magnitudes of \overline{V}_{SSSC} and \overline{I} , respectively. Note that, \overline{I}/I is a unit vector of line current. Therefore, the current \overline{I} in Fig. 2.1, can be expressed as

$$\overline{I} = \frac{\overline{V_1} - \overline{V_2} - jV_{\text{suc}}\overline{I}/I}{jX_L}$$
 (2.2)

where X_L is the reactance of a tie-line, $\overline{V_1}$ and $\overline{V_2}$ are the voltages at buses 1 and 2, respectively. The active power and reactive power flow through bus 1 are

$$P_{12} + jQ_{12} = \overline{V}_1 \overline{I}^* \tag{2.3}$$

where \overline{I}^* is a conjugate of \overline{I} . Substituting \overline{I} from (2.2) into (2.3) yields

$$P_{12} + jQ_{12} = \frac{V_1V_2}{X_L}\sin(\delta_1 - \delta_2) - V_{\infty}\frac{\overline{V_1}\overline{I}^*}{X_LI} + j\left(\frac{V_1^2}{X_L} - \frac{V_1V_2}{X_L}\cos(\delta_1 - \delta_2)\right)$$
 (2.4)

where $\overline{V}_1 = V_1 e^{j\delta_1}$ and $\overline{V}_2 = V_2 e^{j\delta_2}$. In the second term of the right hand side of (2.4), $\overline{V}_1 \overline{I}^*$ is equal to $P_{12} + jQ_{12}$ (see (3.3)). Accordingly, the relation in the real part of (2.4) provides

$$P_{12} = \frac{V_1 V_2}{X_I} \sin(\delta_1 - \delta_2) - \frac{P_{12}}{X_I I} V_{xxx}$$
 (2.5)

In (2.5), the second term of the right hand side is the active power controlled by SSSC. Here, it is assumed that V_1 and V_2 are constant and the initial value of V_{acc} is zero, i.e. $V_{\text{max},0} = 0$. By linearizing (2.5) about an initial operating point,

$$\Delta P_{12} = \frac{V_1 V_2 \cos(\delta_{10} - \delta_{20})}{X_t} (\Delta \delta_1 - \Delta \delta_2) - \frac{P_{120}}{X_t I_1} \Delta V_{00}$$
 (2.6)

where the subscript "0" denotes the value at the initial operating point. As the voltage deviation of SSSC (ΔV_{sssc}) is adjusted, the power output deviation injected by SSSC can be controlled as $\Delta P_{xxsc} = -(P_{120}/X_II_0)\Delta V_{sssc}$. Equation (2.6), therefore, implies that the SSSC is capable of controlling the active power independently. Here, the SSSC is represented by the active power controller. The control effect by SSSC is expressed by the injected power deviation ΔP_{xxsc} instead of $-(P_{120}/X_II_0)\Delta V_{sssc}$. As a result, (2.6) can be expressed as

$$\Delta P_{12} = \Delta P_{T12} + \Delta P_{sssc} \qquad (2.7)$$

where

$$\Delta P_{T12} = \frac{V_1 V_2 \cos(\delta_{10} - \delta_{20})}{X_1} (\Delta \delta_1 - \Delta \delta_2)$$

$$= T_{12} (\Delta \delta_1 - \Delta \delta_2) \qquad (2.8)$$

and T_{ij} is a synchronizing power coefficient.

2.4 Structure of Frequency Stabilizer

In this study, the structure of the frequency stabilizer is based on a second-order lead/lag compensator as shown in Fig. 2.2. There are five parameters for each designed frequency stabilizer consisting of a stabilization gain K, time constants T_1, T_2, T_3 , and T_4 .

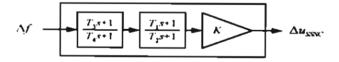


Figure 2.2: Configuration of 2nd order lead/lag based SSSC frequency stabilizer

Since the control purpose of the frequency stabilizer is to enhance the damping of the inter-area mode. The frequency deviation of each target area (Δf_i , i=1 and 3) which provides information of each mode of interest, is used as the input signal for each frequency stabilizer. The control parameters of each frequency stabilizer are searched based on the proposed objective function explained in the following chapter.

Chapter 3

Design of Robust Frequency Stabilizer of SSSC in Continuous Time Domain

3.1 Purpose of this chapter

In this chapter, the robust design method based on continuous time domain of the frequency stabilizer is proposed. First, the problem statement is explained. Next, the design methodology, that is a concept of coordinated control of SSSC and governor, a linearized model of power system, a model reduction by overlapping decompositions technique, a tabu search algorithm for solving optimization problem are described. Subsequently, the effects of the designed frequency stabilizer are evaluated in the four-area interconnected power systems.

3.2 Problem Statement

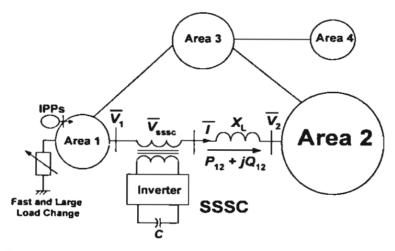


Figure 3.1: An SSSC in a four-area interconnected power system

Figure 3.1 shows the four-area interconnected power system with a loop configuration. This system is used to explain the motivation of the proposed control design. It is assumed that a large load with rapid change has been installed in an area 1. This load change causes serious frequency oscillations. Moreover, IPPs that do not possess frequency control capabilities are included in an area 1. Under this situation, the governors in an area 1 can not sufficiently provide adequate frequency control. On the contrary, the area 2 has large control capability enough to spare for other areas. Therefore, an area 2 offers a service of

frequency stabilization to an area 1 by using the SSSC. Since SSSC is a series-connected device, the power flow control effect is independent of an installed location. In the proposed design method, the SSSC controller uses the frequency deviation of area 1 as a local signal input. Therefore, the SSSC is placed at the point near an area 1. Note that the SSSC is utilized as the energy transfer device from area 2 to area 1. As the frequency fluctuation in an area 1 occurs, the SSSC will provide the dynamic control of a tie line power via the system connections. By exploiting the system interconnections as the control channels, the frequency oscillations can be stabilized.

Additionally, in the interconnected system, variations of system parameters, various load changes etc. cause several system uncertainties. To achieve the high robust stability of system, the effect of uncertainties should be taken into account in the design process. In this study, the aim of the proposed frequency stabilizer design is not only to enhance the damping of interested inter-area oscillation modes, but also to improve the robust stability of system against uncertainties.

3.3 Design Methodology

3.3.1 Coordinated Control of SSSC and Governor

The response of SSSC is extremely rapid when compared to the conventional frequency control system, i.e. a governor. The difference in responses signifies that the SSSC and governor can be coordinated. When a power system is subjected to a sudden load disturbance, the SSSC quickly acts to damp frequency oscillation in the transient period. Subsequently, the governor continues to eliminate the steady-state error in frequency oscillation. As the periods of operation for the SSSC and governor do not overlap, the dynamic of governor can then be neglected in the design of frequency stabilizer for the sake of simplicity.

3.3.2 Linearized Power System Model

A study system depicted in Fig. 3.1 is used to explain the proposed control design of SSSC. The linearized four-area interconnected system including the active power model of SSSC is delineated in Fig.3.2. Based on the coordinated control of SSSC and governors, the dynamics of governors are eliminated in this figure. The active power controller of SSSC has a structure of the lead/lag compensator $K_{RB}(s)$ with an output signal ΔP_{ref} . In this study, the dynamic characteristic of SSSC is modeled as the first order controller with time constant

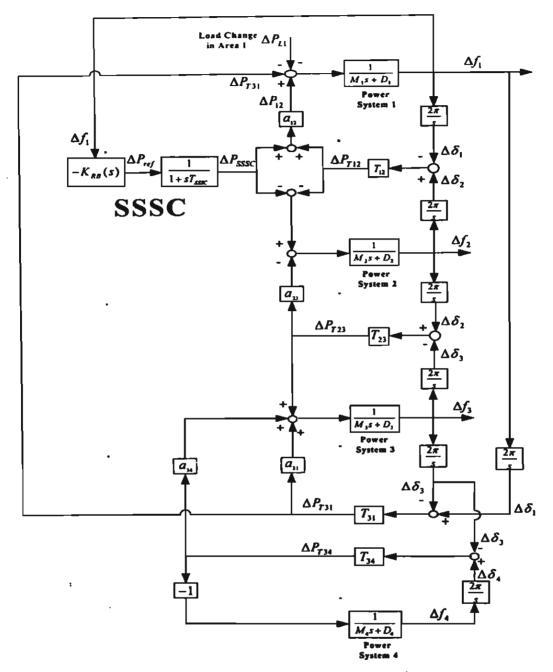


Figure 3.2: An SSSC in a linearized four-area interconnected system without governors

 $T_{\rm SSSC}$. Note that the injected power deviation of SSSC, $\Delta P_{\rm SSSC}$ acting positively on the area 1 reacts negatively on the area 2. Therefore, $\Delta P_{\rm SSSC}$ flows into both areas with different signs (+, -), simultaneously. This characteristic represents the physical meaning of (2.7). The linearized system in Fig.3.2 can be expressed as

$$S:\begin{bmatrix} \Delta f_{1} \\ \Delta \dot{P}_{112} \\ \Delta \dot{f}_{2} \\ \Delta \dot{f}_{2} \\ \Delta \dot{f}_{3} \\ \Delta \dot{f}_{3} \\ \Delta \dot{f}_{3} \\ \Delta \dot{f}_{4} \end{bmatrix} = \begin{bmatrix} -\frac{D_{1}}{M_{1}} & a_{S12} & 0 & a_{S14} & 0 & 0 & 0 \\ -2\pi T_{12} & 0 & 2\pi T_{12} & 0 & 0 & 0 & 0 \\ 0 & -\frac{1}{M_{1}} & -\frac{D_{1}}{M_{2}} & -\frac{a_{21}}{M_{2}} & 0 & 0 & 0 \\ 0 & 0 & 2\pi T_{23} & 0 & -2\pi T_{23} & 0 & 0 \\ 0 & 0 & 0 & 0 & -2\pi T_{24} & 0 & 2\pi T_{34} \\ 0 & 0 & 0 & 0 & 0 & -2\pi T_{34} & 0 \\ 0 & 0 & 0 & 0 & 0 & -2\pi T_{34} & -\frac{D_{4}}{M_{4}} \end{bmatrix} \begin{pmatrix} \Delta f_{1} \\ \Delta P_{712} \\ \Delta f_{2} \\ \Delta f_{3} \\ \Delta f_{3} \\ \Delta f_{4} \end{bmatrix} + \begin{pmatrix} \frac{a_{11}}{M_{1}} \\ 0 \\ -\frac{1}{M_{2}} \\ 0 \\ 0 \\ 0 \\ 0 \end{pmatrix} \Delta P_{xxx} (3.1)$$

 Δf_i is the frequency deviation of area i, ΔP_{Tij} is the power deviation between areas i and j, M_i , is the inertia constant of area i, D_i is the damping coefficient of area i, a_{ij} is the area capacity ratio between areas i and j, T_{ij} is the synchronizing power coefficient of the tie-line between areas i and j, where i, j = 1,...,3. Here $a_{Si2} = (a_{12} + T_{31}/T_{12})/M_1$, $a_{Si4} = -T_{31}/(M_1T_{23})$, $a_{Si2} = -a_{31}T_{31}/(M_3T_{12})$, $a_{Si3} = (1 + a_{31}T_{31}/T_{23})/M_3$. The variable ΔP_{Ti2} is represented in terms of ΔP_{Ti2} and ΔP_{Ti2} by

$$\Delta P_{T31} = -\frac{T_{31}}{T_{12}} \Delta P_{T12} + \frac{T_{31}}{T_{23}} \Delta P_{T23}$$
 (3.2)

Thus, ΔP_{T31} has disappeared in (3.1). This system has three conjugate pairs of complex eigenvalues and a negative real eigenvalue. The former corresponds to three inter-area oscillation modes, and the latter the inertia center mode. In this paper, the design purpose of SSSC is to enhance the damping of the inter-area mode between areas 1 and 2.

3.3.3 Model Reduction by Overlapping Decompositions [32]

The concept of overlapping decompositions is applied to the system (3.1) with the aim of extracting the subsystem where the inter-area mode between areas 1 and 2 is preserved. The system (3.1) is referred to as the system S. The state variables of S are classified into three groups, i.e. $x_1 = [\Delta f_1]$, $x_2 = [\Delta P_{T12}]$, $x_3 = [\Delta f_2, \Delta P_{T23}, \Delta f_3, \Delta P_{T34}, \Delta f_4]^T$. Therefore, the system S can be expressed in compact form as

$$S: \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_1 \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} B_{11} \\ B_{21} \\ B_{31} \end{bmatrix} \Delta P_{SSSC}$$
(3.3)

The sub-matrices A_{ij} and B_{i1} , (i, j = 1, 2, 3) have appropriate dimensions identical to the corresponding state and input vectors. According to the process of overlapping decompositions, the system S can be expanded as

$$\tilde{S} : \begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} & 0 & A_{13} \\ A_{21} & A_{22} & 0 & A_{23} \\ A_{21} & 0 & A_{22} & A_{23} \\ A_{11} & 0 & A_{12} & A_{13} \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \end{bmatrix} + \begin{bmatrix} B_{11} \\ B_{21} \\ B_{21} \\ B_{31} \end{bmatrix} \Delta P_{SSSC}$$
(3.4)

where $z_1 = \left[x_1^T, x_2^T\right]^T$ and $z_2 = \left[x_2^T, x_3^T\right]^T$. The system \bar{S} in (3.4) can be decomposed into two interconnected overlapping subsystems,

$$\tilde{S}_{1} : \dot{z}_{1} = \left(\begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} z_{1} + \begin{bmatrix} B_{11} \\ B_{21} \end{bmatrix} \Delta P_{SSSC} \right) + \begin{bmatrix} 0 & A_{13} \\ 0 & A_{23} \end{bmatrix} z_{2}$$
(3.5)

$$\tilde{S}_{2} : \dot{z}_{2} = \left(\begin{bmatrix} A_{22} & A_{23} \\ A_{32} & A_{33} \end{bmatrix} z_{2} \right) + \begin{bmatrix} A_{21} & 0 \\ A_{31} & 0 \end{bmatrix} z_{1} + \begin{bmatrix} B_{21} \\ B_{31} \end{bmatrix} \Delta P_{SSSC}$$
(3.6)

The state variable x_2 , i.e. the tie line power deviation between areas 1 and 2 (ΔP_{712}) , is repeatedly included in both subsystems, which implies "Overlapping Decompositions".

For system stabilization, consider two interconnected subsystems \tilde{S}_1 and \tilde{S}_2 . The terms in the right hand sides of (3.5) and (3.6) can be separated into the decoupled subsystems (as indicated in the parenthesis in (3.5) and (3.6)) and the interconnection subsystems. As mentioned in [32], if each decoupled subsystem can be stabilized by its own input, the asymptotic stability of the interconnected overlapping subsystems \bar{S}_1 and \tilde{S}_2 are maintained. Moreover, the asymptotic stability of the original system S is also guaranteed. Consequently, the interactions of the decoupled subsystems with the interconnection subsystems in (3.5) and (3.6) are regarded as perturbations. They can be neglected during control design. As a result, the decoupled subsystems of \tilde{S}_1 and \tilde{S}_2 can be expressed as

$$\tilde{S}_{D1} : \dot{z}_{1} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} z_{1} + \begin{bmatrix} B_{11} \\ B_{21} \end{bmatrix} \Delta P_{SSSC}$$
(3.7)

$$\tilde{S}_{D2}$$
 : $\dot{z}_2 = \begin{bmatrix} A_{22} & A_{23} \\ A_{32} & A_{33} \end{bmatrix} z_2$ (3.8)

In (3.7) and (3.8), there is a control input ΔP_{SSSC} appearing only in the subsystem \tilde{S}_{D1} . Here, the decoupled subsystem \tilde{S}_{D1} is regarded as the designed system, which can be expressed as

$$G: \begin{bmatrix} \Delta \dot{f}_1 \\ \Delta \dot{P}_{112} \end{bmatrix} = \begin{bmatrix} -D_1/M_1 & a_{S12} \\ -2\pi T_{12} & 0 \end{bmatrix} \begin{bmatrix} \Delta f_1 \\ \Delta P_{T12} \end{bmatrix} + \begin{bmatrix} a_{12}/M_1 \\ 0 \end{bmatrix} \Delta P_{XXXI}. \tag{3.9}$$

It can be verified that the eigenvalues of (3.9) are complex conjugate and are assumed to be $-\sigma \pm j\omega_d$. These complex eigenvalues correspond to the inter-area oscillation mode between areas 1 and 2 in the original system S. By virtue of overlapping decompositions, the physical characteristic of the original system S is still preserved after the process of system reduction. This explicitly shows the merit of overlapping decompositions.

By incorporating the dynamic characteristic of the SSSC, (3.9) becomes

$$G : \begin{bmatrix} \Delta f_1 \\ \Delta \dot{P}_{112} \\ \Delta \dot{P}_{155C} \end{bmatrix} = \begin{bmatrix} -D_1/M_1 \, a_{S12} \, a_{12}/M_1 \\ -2\pi T_{12} \, 0 \, 0 \\ 0 \, 0 \, -1/T_{SSSC} \end{bmatrix} \begin{bmatrix} \Delta f_1 \\ \Delta P_{T12} \\ \Delta P_{sssc} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1/T_{SSSC} \end{bmatrix} \Delta P_{ref}$$
 (3.10)

For the input signal of SSSC controller, two available local signals, i.e., area 1 frequency deviation (Δf_1) and the tie line 1-2 power deviation (ΔP_{T12}) are taken into consideration. By calculating the right eigenvector of the oscillation mode between areas 1 and 2, the degree of activity [12] of Δf_1 and ΔP_{T12} in this mode can be evaluated. From (3.9), the eigenvalues representing the oscillation mode are $\lambda_{1,2} = -0.015 \pm j3.5316$. The magnitudes of elements of the right-eigenvectors that correspond to Δf_1 and ΔP_{T12} are 0.6088 and 0.1953, respectively. As a result, Δf_1 provides higher degree of activity in this mode. Consequently, Δf_1 is used as the input signal of SSSC controller. The negative feedback control scheme of SSSC controller can be expressed by

$$\Delta P_{ref} = -K_{RB}(s)\Delta f_1 \tag{3.11}$$

The robust controller $K_{RB}(s)$ is in form of a lead/lag stabilizer as

$$K_{RB}(s) = K \frac{T_w s}{1 + T_w s} \frac{(1 + T_1 s)}{(1 + T_2 s)} \frac{(1 + T_3 s)}{(1 + T_4 s)}$$
(3.12)

where

K: a controller gain

 T_1 , T_2 , T_3 , T_4 : lead/lag time constants [sec]

 T_{\bullet} : a washout time constant [sec]

Here, T_{w} is set to 10 [sec]. The control parameters K, T_{1}, T_{2}, T_{3} , and T_{4} are searched based on the objective function explained in the next section.

3.3.4 Determination of Objective Function

In derivation of the objective function, both attenuation performance of system disturbances, and robust stability of controller against system uncertainties are taken into consideration. Since the main purpose of SSSC is to limit the peak frequency deviation following a sudden load perturbation, the peak frequency deviation can be used as a design specification. Assume that the eigenvalues corresponding to the mode of frequency oscillation in the uncontrolled system G are determined as $-\sigma \pm j\omega_d$. Thus, the system peak response to the unit step input [33] is given by

$$M_{P(actual)} = 1 + \exp(-\sigma \pi/\omega_d)$$
 (3.13)

If the peak allowable frequency deviation of the controlled system is specified to be $M_{P(design)}$, then the magnitude of the difference between the design and the actual peak frequency deviations can be defined as

$$\psi = \left| M_{P(design)} - M_{P(actual)} \right| \tag{3.14}$$

This is the part of disturbance attenuation performance in the objective function that will be minimized.

Next, the robust stability against system uncertainties is taken into consideration. The possible uncertainties are ignored nonlinear characteristics of the study system, ranges and bounds for uncertain system parameters etc. Practically, it is hardly to know about the information of all uncertainties existing in the system. To consider such system uncertainties, the unstructured uncertainty can be applied. In general, an upper bound of the magnitude (or size) of unstructured uncertainty can be estimated. If the magnitude of system uncertainties is less than this upper bound, the robust stability of system is guaranteed. Here, the multiplicative uncertainty [33-25] is applied to represent the unstructured uncertainty in the system, as shown in Fig.3.3. Note that \tilde{G} is a nominal system transfer function, Δ_m is a stable multiplicative perturbation, and \tilde{K} is a controller designed to ensure the internal stability of the nominal closed loop.

Based on the small-gain theorem [33-35], the closed loop system will be robustly stable if

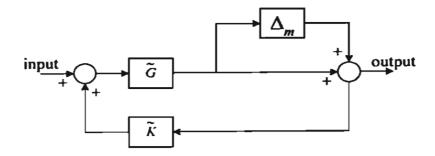


Figure 3.3: Feedback system with multiplicative uncertainty

$$\left|\Delta_{m}\right| < \frac{1}{\left|\tilde{G}\tilde{K}(1+\tilde{G}\tilde{K})^{-1}\right|} \tag{3.15}$$

where, the symbol $|\Delta_m|$ shows the magnitude of uncertainty. $|\bar{G}\tilde{K}(1+\bar{G}\bar{K})^{-1}|$ is the magnitude of complementary sensitivity function which is referred to as |T|.

As mentioned in [33-35], the multiplicative stability margin (MSM) is defined as

$$MSM = 1/\|T\|_{\infty} \tag{3.16}$$

where $||T||_{\infty}$ is the ∞ -norm of T. The MSM can be used to measure the robust stability of the system. Large value of MSM exhibits high robust stability of the system.

From (3.15) and (3.16), it is cleared that if the controller \tilde{K} can be designed to minimize $\|T\|_{\infty}$, the MSM increases. As a result, the upper bound of $|\Delta_m|$ is enlarged, and the high robust stability will be ensured. Thus, the robustness index in the objective function can be defined in normalized form as,

$$\gamma = \|T\|_{\infty} / \|T\|_{\infty(initial)} \tag{3.17}$$

where $\|T\|_{\infty(sum not)}$ is the ∞ -norm of T at the initial solution of search process.

Combining (3.14) and (3.17), the objective function F can be formulated as,

Minimize
$$F = c \cdot \psi + \gamma$$

Subject to $K_{\min} \leq K \leq K_{\max}$. (3.18)
 $T_{i,\min} \leq T_{i} \leq T_{i,\max}$

The constant coefficient "c" is used to weight ψ -term, so that c- ψ dominates γ during the parameters optimization. Note that, since γ is normalized to 1 at the initial

solution, it is easy to find the value of "c" so that $c \cdot \psi$ is greater than 1. Eventually, the search process minimizes both terms until $c \cdot \psi$ meets the design specification and γ decreases to the possible minimum value. The minimum and maximum values of the gain K are set as 0.1 and 500, respectively. The minimum and maximum values of the time constants T_i (i = 1, 2, 3, 4) are set as 0.01 and 5, respectively. In this research, tabu searach algorithm (TSA) is employed to solve this optimization problem and search for optimal parameters of controller.

3.3.5 TSA for Parameter Determination

The TSA [36-37] is a promising tool for solving combinatorial optimization problem. The algorithm is an iterative improvement procedure that can start from any initial solution. Three basic components of TSA are used as follows: the trial solution generation, tabu list (TL) restriction, and termination criterion.

The concatenated encoding method is employed as shown in Fig. 3.4.

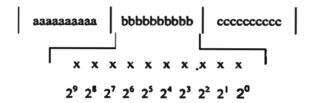


Figure 3.4: Example of a 30 bit concatenated encoding scheme

Each parameter is encoded in a binary string normalized over its range. This encoding method stacks each normalized string in series with each other to construct the string individual. The same number of n bits is used to represent each parameter string.

To obtain the actual value of each parameter, (3.19) is used to decode each normalized string to its decimal value for objective function evaluation.

$$P_{i} = P_{i,\min} + \frac{B_{i} \times [P_{i,\max} - P_{i,\min}]}{2^{n} - 1}$$
 (3.19)

where P_i is the actual value of the *i*-th parameter, $P_{i,min}$ is the minimum value of the *i*-th parameter, $P_{i,max}$ is the maximum value of the *i*-th parameter, B_i is the decimal integer value of binary string of the *i*-th parameter, and n is the number of bits representing each parameter. In the design process, 10 bits are used to represent each parameter.

To generate a trial solution of an initial feasible solution, one bit of binary string is flipped at a time. The maximum number of trial solutions per iteration is referred to a

neighborhood solution space (NS). In this study, NS is set to 95% of the total number of bits $|0.95 \times n \times N|$, where N is a number of parameter searched.

The example of generating trial solutions is shown in Fig.3.5.

Initial solution: 1001011010 1100011010 1110010011
Trial solution 1: 10001011010 1100011010 1110010011
Trial solution 2: 11101011010 1100011010 1110010011
Trial solution 3: 101101010 1100011010 1110010011

Figure 3.5: Concept of Trial Solution Generation

The table list (TL) is referred to as an adaptive memory. The mechanism of TL is to keep attributes (bit positions) that created the best solution of the past iterations in the TL for a certain period. The attributes included in the TL cannot be used to create new solution candidates as long as they are in the TL. As the iteration proceeds, a new attribute enters into the TL as a fixed attribute. At the same time, the oldest attribute is released from the TL and becomes a free attribute, as illustrated in Fig. 3.6.

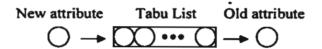


Figure 3.6: Mechanism of tabu list

In particular, a size of TL affects the quality of the solution. It controls the search process to avoid being trapped in local optima. Note that the size of TL or so-called the tabu length, is only the control parameter of TSA. Basically, the tabu length that provided good solutions usually grows with the size of the problem.

However, observing the quality of the solution can identify the appropriate tabu length. If the tabu length is too small, the cycling of solution occurs in the search process. On the other hand, if the size is too large, the search process is too restricted and may deteriorate the solution. Here, the tabu length is set to $\lfloor 0.7 \times n \times N \rfloor$.

Termination criterion refers to the condition that the search process will terminate. In the design, the search will terminate when the number of iterations reaches 100.

To apply the TSA for optimal parameter determination, the initial feasible solution is generated arbitrarily. A move to a neighbor solution is performed if the TL does not restrict

it. The best solution is updated during the search process until the termination criterion is satisfied. The following notations are used for the TSA procedure:

TL: the tabu list.

NS: the neighborhood solution space.

F(X): the objective function of solution X,

 F_h^{λ} : the best objective function at iteration k,

 X_a^k : the initial feasible solution at iteration k,

 X_m^k : a trial m solution at iteration k,

 X_{cb}^{k} : the current best trial solution at iteration k,

 X_h^k : the best solution reached at iteration k,

 k_{max} : the maximum allowable number of iterations.

The tabu search procedure can be described as follows:

1. Read the constraints of searched parameters, the initial feasible solution X_o^k , and specification of the controller.

2. Specify the length of TL, k_{max} , and size of NS.

3. Initialize iteration counter k and empty TL.

4. Set $X_b^{\ k} = X_a^{\ k}$.

5. Execute tabu search procedure:

5.1 Initialize the trial counter m to zero.

5.2 Generate a trial solution X_m^k from X_n^k .

5.3 If X_m^k is not feasible, go to 5.8.

5.4 If X_m^k is the first feasible solution, set $X_{ch}^k = X_m^k$.

5.5 Perform the tabu test. If X_m^k is tabued, then go to 5.8.

5.6 If $F(X_m^k) \le F(X_{ch}^k)$, set $X_{ch}^k = X_m^k$.

5.7 If $F(X_m^k) < F(X_h^k)$, set $X_h^k = X_m^k$.

5.8 If m is less than NS, m = m + 1 and go to 5.2.

5.9 If there is no feasible solution, set $X_o^{k+1} = X_b^k$. Otherwise, set $X_o^{k+1} = X_{cb}^k$, and update TL.

6 If $k \le k_{max}$, then k = k + 1, and go to 5.

7 X_h^A is the best solution found.

3.4 Simulation Results and Evaluation

To determine five control parameters by the TSA, the values of N, NS and the tabu length are set to 5, 47 and 35 respectively. Following the design procedures and appropriately

setting c = 1.4 in the objective function, the following transfer function was obtained for the robust controller of SSSC when the peak frequency deviation was limited to $M_{material} = 1.2$.

$$K_{RR}(s) = 3.5206 \frac{10s}{1+10s} \frac{(1+0.7758s)}{(1+2.5074s)} \frac{(1+1.9660s)}{(1+3.7269s)}$$
 (3.20)

The SSSC controller in (3.20) is compared to the designed SSSC with $M_{p(design)} = 1.2$ but without robustness consideration. By neglecting γ in the objective function and setting c = 1.0, the designed result is given by

$$K_{NRR}(s) = 5.4753 \frac{10s}{1+10s} \frac{(1+0.1271s)}{(1+2.7026s)} \frac{(1+3.8781s)}{(1+2.4440s)}$$
 (3.21)

Note that the designed controllers in (3.20) and (3.21) are referred to as "Robust SSSC" and "Non-robust SSSC" respectively.

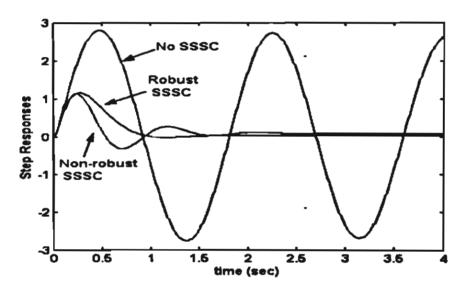


Figure 3.7: System step responses

Figure 3.7 depicts the step responses of G. Without SSSC, the oscillations are undamped with the first peak value about 2.7875. On the other hand, the first peak values are reduced to 1.2025 and 1.1225 in cases of Robust and Non-robust SSSCs, respectively. In addition, the MSM of system G with Robust SSSC is increased to 0.90 from 0.57 in case of system with Non-robust SSSC. This reveals the higher robustness of the system G with Robust SSSC.

To prove that the robust controller designed in the decoupled subsystem (3.7) can guarantee the stability of the expanded system (3.4) and also the original system (3.3), the eigenvalue analysis is employed. Table 3.1 shows the eigenvalues of decoupled subsystems

(3.7) and (3.8). The eigenvalues $\lambda_{1,2}$ of (3.7) represent the inter-area mode between areas 1 and 2. After applying the control input ΔP_{SSSC} , this oscillation mode is stabilized effectively. On the other hand, eigenvalues of (3.8) have not been changed. This is due to no control input ΔP_{SSSC} in (3.8). Table 3.2 shows the eigenvalues of expanded system (3.4). By considering the participation factor [12], it can be verified that $\lambda_{1,2}$ represents the inter-area mode between areas 1 and 2. As expected, ΔP_{SSSC} can stabilize this oscillation mode. For other modes, they are stable after the controller is included. Note that λ_{3} is approximately zero which has no physical meaning. This is because of a redundant state variable ΔP_{712} in (3.4). For the eigenvalues of the original system (3.3) demonstrated in Table 3.3, $\lambda_{1,2}$ can also be stabilized by ΔP_{SSSC} . Other modes are also stable after control. These eigenvalue analysis results confirm the concept of overlapping decompositions that the stability of the original system can be guaranteed if the decoupled subsystems can be stabilized by its own input.

Table 3.1: Eigenvalue Analysis Results of Decoupled Subsystems (3.7) and (3.8)

	Before Control	After Control	
Decoupled Subsystem (3.15)	$\lambda_{1,2} = -0.015 \pm j3.5316$	$\lambda_{1,2} = -3.6077 \pm j3.7967$ $\lambda_{3,4} = -0.1027 \pm j0.0642$ $\lambda_{5} = -0.5020$ $\lambda_{6} = -12.8613$	
Decoupled Subsystem (3.16)	$\lambda_{1,2} = -0.0152 \pm j1.6777$ $\lambda_{3,4} = -0.0168 \pm j3.6922$ $\lambda_{5,6} = -0.0247 \pm 2.1532$	Not change	

Table 3.2: Eigenvalue Analysis Results of Expanded System (3.4) (or interconnected overlapping subsystems (3.5) and (3.6))

	Before Control	After Control
Expanded System (3.12)	$ \frac{\lambda_{1,2} = -0.0171 \pm j4.4134}{\lambda_{3,4} = -0.0184 \pm j3.5602} $ $ \lambda_{5,6} = -0.0157 \pm j1.7520 $ $ \lambda_{7} = -0.0409 $ $ \lambda_{8} = -4.18 \times 10^{-14} $	$ \frac{\lambda_{1.2} = -3.8240 \pm j5.0223}{\lambda_{3.4} = -0.0461 \pm j3.6994} $ $ \lambda_{5.6} = -0.0061 \pm j1.7364 $ $ \lambda_{7} = -0.0414 $ $ \lambda_{8} = -2.84 \times 10^{-14} $ $ \lambda_{9,10} = -0.1232 \pm j0.0483 $ $ \lambda_{11} = -0.4485 $ $ \lambda_{12} = -12.4 $

Table 3.3: Eigenvalue Analysis Results of Original System (3.1)

	Before Control	After Control	
Original System (3.11)	$ \frac{\lambda_{1,2} = -0.0171 \pm j4.4134}{\lambda_{3,4} = -0.0184 \pm j3.5602} $ $ \lambda_{5,6} = -0.0157 \pm j1.7520 $ $ \lambda_{7} = -0.0409 $	$ \frac{\lambda_{1,2} = -3.8240 \pm j5.0223}{\lambda_{3,4} = -0.0461 \pm j3.6994} $ $ \lambda_{5,6} = -0.0061 \pm j1.7364 $ $ \lambda_{7} = -0.0414 $ $ \lambda_{8,9} = -0.1232 \pm j0.0483 $ $ \lambda_{10} = -0.4485 $ $ \lambda_{11} = -12.4217 $	

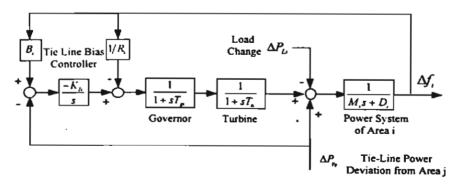


Figure 3.8: Linearized system model of area i including governor

Table 3.4: Data of Four-area Interconnected Power System (Area Capacity Ratio 5:10:2:0.8)

System Parameters	Area 1	Area 2	Area 3	Area 4
Inertia Constant	$M_1 = 0.2$	$M_2 = 0.0167$	$M_3 = 0.15$	$M_4 = 0.2$
(MW.s/Hz)				
Damping Coefficient	$D_1 = 0.006$	$D_2 = 0.00833$	$D_3 = 0.005$	$D_4 = 0.006$
(MW/Hz)				
Turbine Time	$T_{c1} = 0.25$	$T_{12} = 0.3$	$T_{r_3} = 0.25$	$T_{t4} = 0.25$
Constant (s)				
Governor Time	$T_{g1} = 0.1$	$T_{g2}=0.08$	$T_{g3} = 0.1$	$T_{g4} = 0.1$
Constant (s)				
Regulation Ratio	$R_1 = 2.4$	$R_2 = 2.4$	$R_3 = 2.4$	$R_4 = 2.4$
(Hz/MW)			<u> </u>	
Bias Coefficient	$B_{\rm i} = 0.5$	$B_2 = 0.5$	$B_3 = 0.5$	$B_4^{:} = 0.5$
(MW/Hz)				
Integral Controller	$K_{i1} = 0.5$	$K_{i2} = 0.5$	$K_{i3} = 0.5$	$K_{14} = 0.5$
Gain (1/s)				
Synchronizing Power				
Coefficient (MW/rad)	$T_{12} = 0.159$, $T_{23} = 0.064$, $T_{31} = 0.079$, $T_{31} = 0.079$			= 0.079
Area Capacity Ratio	$a_{12} = 2.0$, $a_{23} = 0.2$, $a_{31} = 2.5$, $a_{31} = 0.4$			

Next, the disturbance attenuation performance and robustness of both designed controllers are investigated in the linearized model of the four-area interconnected system. The dynamic of governor [11] is also incorporated into each area, as delineated in Fig. 3.8. System parameters are given in Table 3.4.

In order to evaluate the disturbance attenuation performance of both controllers, a sudden step load of 0.01 [p.u. MW] is applied to area 1 at t = 1.0 [sec]. The frequency deviation of each area is depicted in Figs. 3.9 - 3.12. Without control of the SSSC, the fluctuations of frequency deviations in all areas are large with poor damping. After the inclusion of both controllers, frequency oscillations in all areas are effectively stabilized. Especially, the peak value of frequency deviation in the controlled area 1 is significantly suppressed. In addition, the oscillating shapes are also stabilized completely. Meanwhile, steady-state errors of frequency deviations are eliminated slowly due to the effects of the governors. Furthermore, the tie line 1-2 power deviation illustrated in Fig. 3.13 is also effectively stabilized by Robust SSSC. As shown in Fig. 3.14, the maximum injected power

deviation of Non-robust SSSC is 0.0094 [p.u. MW], which is almost equal to the size of load change. In contrast, due to consideration of both performance and robustness in design process, the injected power deviation of Robust SSSC is 0.0058 [p.u. MW].

Here, the robustness of each designed controller is evaluated. A random load disturbance composed of several oscillation frequencies, $\Delta P_{LI} = 0.002 sin(3t) + 0.005 sin(6t) - 0.007 sin(9t)$ [p.u. MW] is applied to area 1. At the same time, the damping coefficient in area 1 (D_i) is changed from 0.006 [p.u. MW / Hz] (positive damping) to -0.75 [p.u. MW / Hz] (negative damping). Note that the negative damping signifies an unstable system operation. As clearly illustrated by the area 1 frequency oscillations in Fig. 3.15, the system completely loses stability in cases of a Non-robust SSSC and no SSSC. Moreover, frequency oscillations in other areas (not shown here) severely fluctuate and finally diverge. On the other hand, the Robust SSSC explicitly maintains its performance against large uncertainties and severe perturbations. Frequency oscillations in area 1 and other areas are perfectly stabilized. The efficiency of the proposed SSSC is also evident in Figs. 3.16 and 3.17, where the power deviation in the tie line 1-2 and the injected power deviation of SSSC are shown.

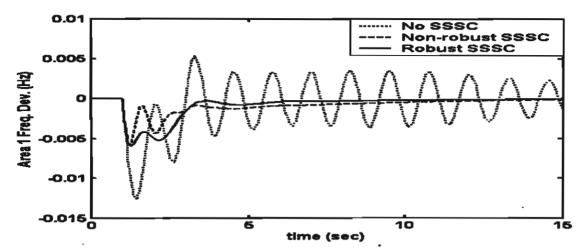


Figure 3.9: Frequency deviation of area 1

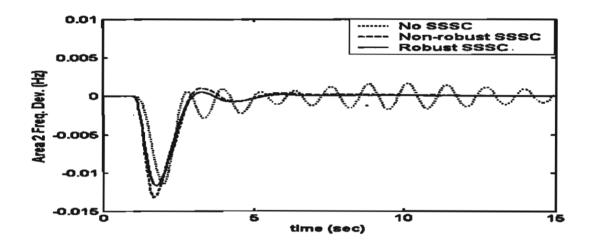


Figure 10: Frequency deviation of area 2

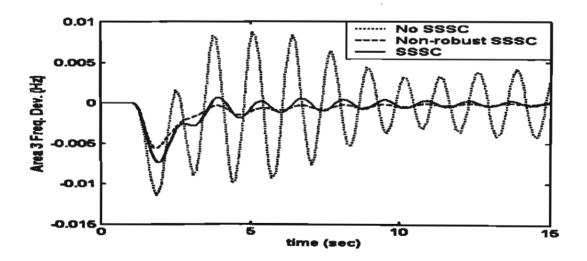


Figure 3.11: Frequency deviation of area 3

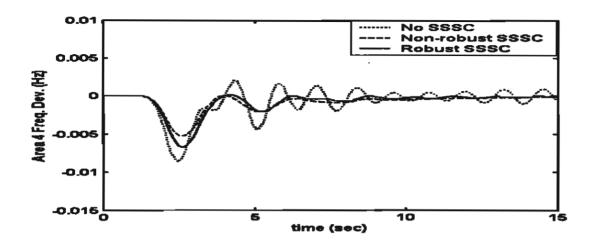


Figure 3.12: Frequency deviation of area 4 '

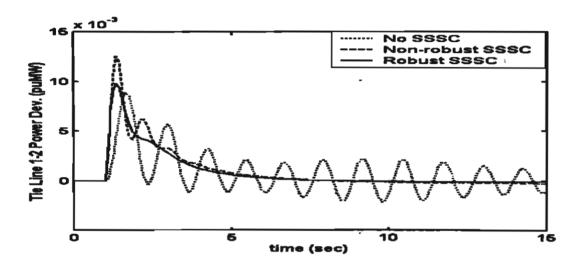


Figure 3.13: Tie line 1-2 power deviation

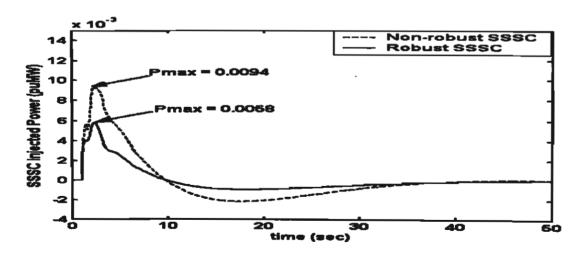


Figure 3.14: Injected power deviation of SSSC

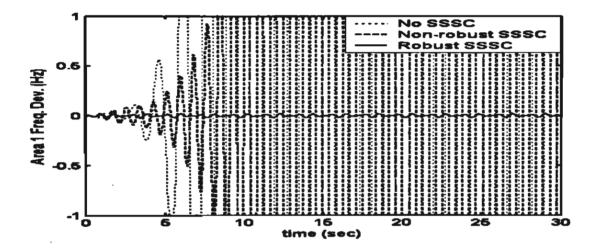


Figure 3.15: Frequency deviation of area 1

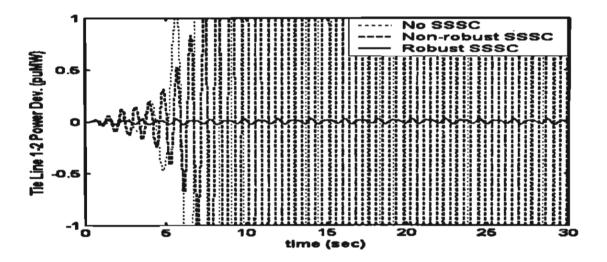


Figure 3.16: Tie line 1-2 power deviation

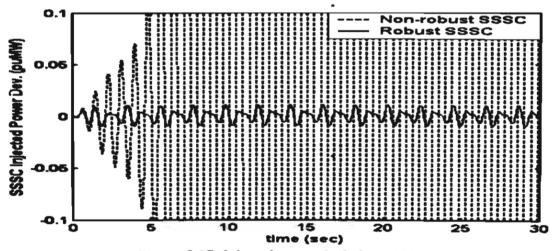


Figure 3.17: Injected power deviation of SSSC

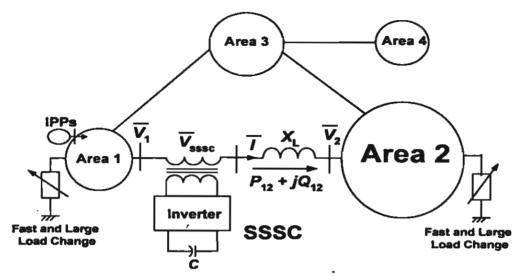


Figure 3.18: An additional load change in area 2

Finally, the performances of both SSSC controllers are evaluated when the load change occurs in area 2. Figure 3.18 shows an additional load change in area 2. It is assumed that a sudden step load of 0.01 [MW] occurs in area 2 at t = 1.0 [sec]. As depicted in Fig. 3.19, both Non-robust SSSC and Robust SSSC are capable of stabilizing frequency oscillations of area 1, even though a load disturbance occurs in area 2. For frequency deviations in other areas (Figs. 3.20-3.22) and tie line 1-2 power deviation (Fig. 3.23), the performance of Robust SSSC is comparatively better than that of Non-robust SSSC. As illustrated in Fig. 3.24, the peak values of injected power deviations of Robust SSSC and Non-robust SSSC are 0.01 [p.u. MW] and 0.027 [p.u. MW], respectively. This implies that the required MW capacity of robust SSSC is lower than that of Non-robust SSSC.

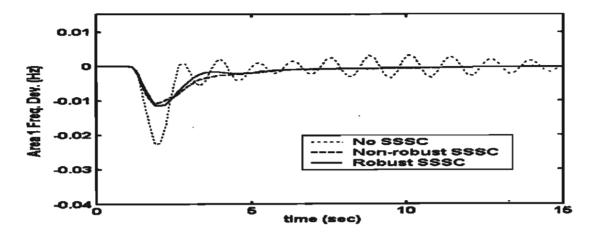


Figure 3.19: Area 1 frequency deviation

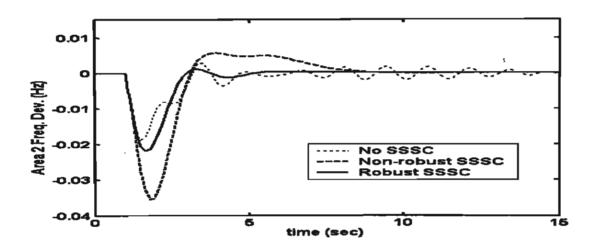


Figure 3.20: Area 2 frequency deviation

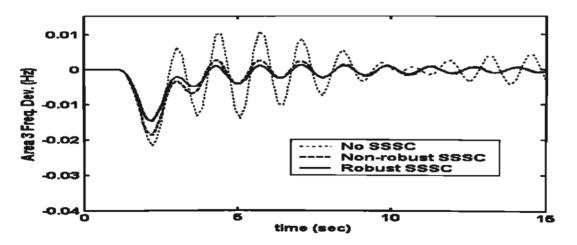


Figure 3.21: Area 3 frequency deviation

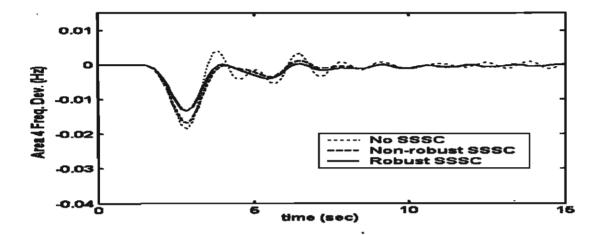


Figure 3.22: Area 4 frequency deviation

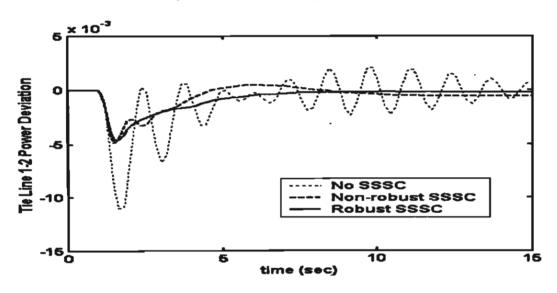


Figure 3.23: Tie line 1-2 power deviation

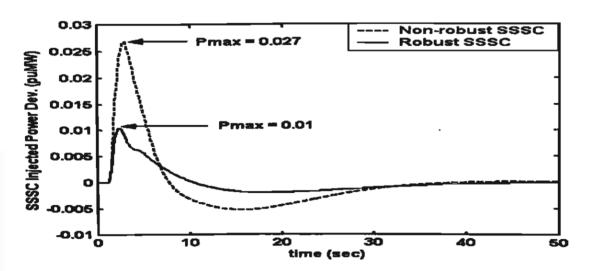


Figure 3.24: SSSC injected power deviation

3.5 Conclusions

This chapter presents the robust design of the lead/lag frequency stabilizer in the continuous time domain. The tabu search algorithm was employed to achieve the optimal parameters. By virtue of the objective function that limits the peak frequency deviation and maximizes the robust stability margin, the proposed design guarantees both performance and robustness of the resulted frequency stabilizer. More specifically, the designed robust frequency stabilizer uses only the frequency deviation of the controlled area as the feedback input signal. This allows practical realization and implementation in a power system. Simulation results clearly demonstrate the superior robustness and performance of the frequency stabilizer designed by the proposed method.

Chapter 4

Design of Robust Frequency Stabilizer of SSSC in Discrete Time Domain

4.1 Purpose of this chapter

Many of today's control systems use digital computers to provide the compensation. In this chapter, to implement the frequency stabilizer in real system, the proposed robust design is applied to establish a discrete-based frequency stabilizer of SSSC. First, the continuous-based frequency stabilizer is designed by the proposed method in Chapter 3. Subsequently, the resulted continuous-based frequency stabilizer is transformed to the discrete-based frequency stabilizer via a digital control technique. In addition, the proposed method is applied to a design problem in case of many frequency stabilizers of SSSCs are installed in an interconnected power system. Simulation study in a three-area loop system shows the significant effects of the designed frequency stabilizers against load changes and negative damping.

4.2 Problem Statement

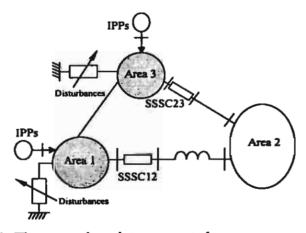


Figure 4.1: Three-area loop interconnected power system with SSSC

A three-area loop interconnected power system depicted in Fig.4.1 is used to explain the motivation of the proposed frequency stabilizer design. SSSC12 and SSSC23 are connected in series with tie-lines between areas 1 and 2, and areas 2 and 3, respectively. It is assumed that large loads with changing frequency in the vicinity of the inter-area oscillation

cause frequency and tie-line power oscillations. In addition, Independent Power Producers (IPPs) that do not possess frequency control capabilities are also included in areas 1 and 3. Thus, it becomes beyond the abilities of governors in areas 1 and 3 to provide adequate frequency controls. Therefore, both SSSCs are applied to utilize the control capabilities of area 2 to compensate for fast load changes in areas 1 and 3. Moreover, by utilizing the area interconnections as the control channels of dynamic power flow control of SSSCs, the frequency oscillations in areas 1 and 3, as well as tie-line power oscillations due to inter-area modes can be effectively stabilized.

Additionally, in the interconnected system, variations of system parameters, various load changes etc. cause several system uncertainties. To achieve the high robust stability of system, the effect of uncertainties should be taken into account in the design process. In this study, the aim of the proposed frequency stabilizer design is not only to enhance the damping of interested inter-area oscillation modes, but also to improve the robust stability of system against uncertainties.

4.3 Design Methodology

4.3.1 Coordinated Control of SSSC and Governor

The response of SSSC is extremely rapid when compared to the conventional frequency control system, i.e. a governor. The difference in responses signifies that the SSSC and governor can be coordinated. When a power system is subjected to a sudden load disturbance, the SSSC quickly acts to damp frequency oscillation in the transient period. Subsequently, the governor continues to eliminate the steady-state error in frequency oscillation. As the periods of operation for the SSSC and governor do not overlap, the dynamic of governor can then be neglected in the design of frequency stabilizer for the sake of simplicity.

4.3.2 Linearized Power System Model

The power system shown in Fig. 4.1 can be represented by a linearized power system model, as shown in Fig. 4.2. Note that the governors are eliminated in this system. The SSSC model is represented by the active power flow controller. The dynamic characteristic of SSSC is modeled as the first order controller with a time constant $T_{SSSC} = 0.05$ sec. The designed discrete frequency stabilizers of SSSC12 and SSSC23 are represented by $K_{12}(z)$ and $K_{23}(z)$, respectively. The discrete frequency stabilizer consists of three main components, i.e., an analog-to-digital converter (ADC), a discrete controller (K(z)) and a digital-to-analog converter (DAC).

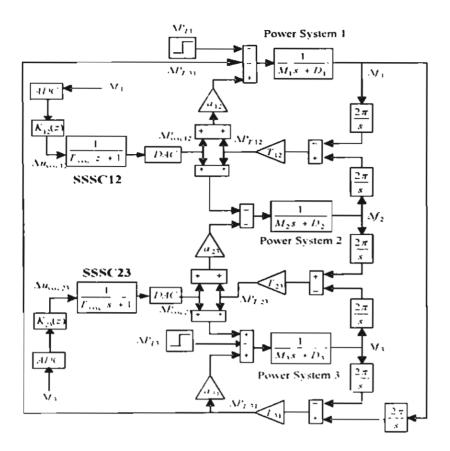


Figure 4.2: Discrete-based frequency stabilizers of SSSCs in a linearized power system model without governors

Usually, a continuous error signal, i.e. a frequency deviation (Δf) is digitized by an ADC. The on-board computer processes this signal to provide a control signal to the plant. Because most plants are continuous, the computer's output is passed through a DAC. The injected power deviation of each SSSC ($\Delta P_{SSSC-12}$ and $\Delta P_{SSSC-23}$) acting positively on an area reacts negatively on another area. Thus, each injected power flows into both areas with different signs (+,-), simultaneously. By neglecting T_{SSSC} , the linearized state equation of Fig. 4.2 can be expressed as

$$\Delta x = A\Delta x + B\Delta u \tag{4.1}$$

where

$$A = \begin{bmatrix} -D_1/M_1 & a_{S12} & 0 & a_{S14} & 0 \\ -2\pi T_{12} & 0 & 2\pi T_{12} & 0 & 0 \\ 0 & -1/M_2 & -D_2/M_2 & -a_{23}/M_2 & 0 \\ 0 & 0 & 2\pi T_{23} & 0 & -2\pi T_{23} \\ 0 & a_{S52} & 0 & a_{S54} & -D_3/M_3 \end{bmatrix},$$

$$B = \begin{bmatrix} a_{12}/M_1 & 0 \\ 0 & 0 \\ -1/M_2 & -a_{23}/M_2 \\ 0 & 1/M_3 \\ 0 & 0 \end{bmatrix},$$

$$\Delta x = \begin{bmatrix} \Delta f_1 & \Delta P_{T12} & \Delta f_2 & \Delta P_{T23} & \Delta f_3 \end{bmatrix}^T,$$

$$\Delta u = \begin{bmatrix} \Delta P_{SSSC12} & \Delta P_{SSSC23} \end{bmatrix}^T.$$

 Δf_i is the frequency deviation of area i, ΔP_{7ij} is the power deviation between areas i and j, M_i is the inertia constant of area i, D_i is the damping coefficient of area i, a_{ij} is the area capacity ratio between areas i and j, T_{ij} is the synchronizing power coefficient of the tie-line between areas i and j, where i, j = 1,...,3. Here $a_{S12} = (a_{12} + T_{3i}/T_{12})/M_1$, $a_{S14} = -T_{3i}/(M_1T_{23})$, $a_{S52} = -a_{3i}T_{3i}/(M_3T_{12})$, $a_{S54} = (1 + a_{3i}T_{3i}/T_{23})/M_3$. The variable ΔP_{731} is represented in terms of ΔP_{712} and ΔP_{723} by

$$\Delta P_{T31} = -\frac{T_{31}}{T_{12}} \Delta P_{T12} + \frac{T_{31}}{T_{23}} \Delta P_{T23}$$
 (4.2)

Thus, ΔP_{T31} has disappeared in (4.1). This system has two conjugate pairs of complex eigenvalues and a negative real eigenvalue. The former corresponds to two inter-area oscillation modes, and the latter the inertia center mode. Based on the mode controllability matrix [12], the inter-area modes between areas 1 and 2, and areas 2 and 3 are controllable for the control inputs Δu_{SSSC12} and Δu_{SSSC23} , respectively. Accordingly, the design purpose of frequency stabilizer is to enhance the damping of the mentioned inter-area modes.

4.3.3 Model Reduction by Overlapping Decompositions

The technique of overlapping decompositions is applied to reduce the system (4.1) to a subsystem embedded with only the inter-area mode of interest. The original system (4.1) is referred to as the system S.

$$S: \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} & A_{13} \\ A_{21} & A_{22} & A_{23} \\ A_{31} & A_{32} & A_{33} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} B_{11} & B_{12} \\ B_{21} & B_{22} \\ B_{31} & B_{32} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$
(4.3)

The sub-matrices A_{ij} and B_{ij} , (i, j = 1, 2, 3) have appropriate dimensions identical to the corresponding states and input vectors. According to the process of overlapping decompositions, the system S can be expressed as

$$\bar{S} : \begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} & 0 & A_{13} \\ A_{21} & A_{22} & 0 & A_{23} \\ A_{21} & 0 & A_{22} & A_{23} \\ A_{31} & 0 & A_{32} & A_{33} \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \end{bmatrix} + \begin{bmatrix} B_{11} \\ B_{21} \\ B_{21} \\ B_{31} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$(4.4)$$

where $z_1 = \begin{bmatrix} x_1^T, x_2^T \end{bmatrix}^T$ and $z_2 = \begin{bmatrix} x_2^T, x_3^T \end{bmatrix}^T$. Subsequently, the system \tilde{S} can be decomposed into two interconnected overlapping subsystems, i.e.

$$\tilde{S}_{1} : \dot{z}_{1} = \left(\begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} z_{1} + \begin{bmatrix} B_{11} \\ B_{21} \end{bmatrix} u_{1} \right) + \begin{bmatrix} 0 & A_{13} \\ 0 & A_{23} \end{bmatrix} z_{2} + \begin{bmatrix} B_{12} \\ B_{22} \end{bmatrix} u_{2}$$
(4.5)

and

$$\tilde{S}_{2} : \dot{z}_{2} = \left(\begin{bmatrix} A_{22} & A_{23} \\ A_{32} & A_{33} \end{bmatrix} z_{2} + \begin{bmatrix} B_{21} \\ B_{31} \end{bmatrix} u_{2} \right) + \begin{bmatrix} A_{21} & 0 \\ A_{31} & 0 \end{bmatrix} z_{1} + \begin{bmatrix} B_{21} \\ B_{31} \end{bmatrix} u_{1}$$
(4.6)

The state variable x_2 is repeatedly included in both subsystems, which implies "Overlapping Decompositions".

For system stabilization, two interconnected subsystems \tilde{S}_1 and \tilde{S}_2 are considered. The terms in the right hand sides of (4.5) and (4.6) can be separated into the decoupled subsystems (as indicated in the parenthesis in (4.5) and (4.6)) and the interconnected subsystems. As mentioned in [32], if each decoupled subsystem can be stabilized by its own input, the asymptotic stability of the interconnected overlapping subsystem \tilde{S}_1 and \tilde{S}_2 are maintained. Moreover, the asymptotic stability of the original system S is also guaranteed. Consequently, the interactions of the decoupled subsystems with the interconnection subsystems in (4.5) and (4.6) are regarded as perturbations. They can be neglected during control design. As a result, the decoupled subsystems of (4.5) and (4.6) can be expressed as

$$\tilde{S}_{D1}$$
: $\dot{z}_{1} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} z_{1} + \begin{bmatrix} B_{11} \\ B_{21} \end{bmatrix} u_{1}$ (4.7)

and

$$\tilde{S}_{D2}$$
 : $\dot{z}_2 = \begin{bmatrix} A_{22} & A_{23} \\ A_{32} & A_{33} \end{bmatrix} z_2 + \begin{bmatrix} B_{22} \\ B_{32} \end{bmatrix} u_2$ (4.8)

By regarding the power deviation between areas 1 and 2 (ΔP_{712}) as the overlapped variable for design of frequency stabilizer of SSSC12, the subsystem embedded with the inter-area mode between areas 1 and 2 can be expressed as

$$\hat{G}_{s1} : \begin{bmatrix} \Delta \hat{f}_1 \\ \Delta \hat{P}_{r12} \end{bmatrix} = \begin{bmatrix} -D_1/M_1 & a_{s12} \\ -2\pi T_{12} & 0 \end{bmatrix} \begin{bmatrix} \Delta \hat{f}_1 \\ \Delta \hat{P}_{r12} \end{bmatrix} + \begin{bmatrix} a_{12}/M_1 \\ 0 \end{bmatrix} \Delta P_{SSSC12}$$
(4.9)

Next, by considering the power deviation between areas 2 and 3 (ΔP_{721}) as the overlapped variable for design of frequency stabilizer of SSSC23, the subsystem embedded with the inter-area mode between areas 2 and 3 can be expressed as

$$\tilde{G}_{s2}:\begin{bmatrix}\Delta\dot{P}_{723}\\\Delta\dot{f}_{3}\end{bmatrix}=\begin{bmatrix}0&-2\pi T_{23}\\a_{s54}&-D_{3}/M_{3}\end{bmatrix}\begin{bmatrix}\Delta P_{723}\\\Delta f_{3}\end{bmatrix}+\begin{bmatrix}0\\1/M_{3}\end{bmatrix}\Delta P_{SSSC23} \qquad (4.10)$$

By incorporating the dynamic characteristic of each SSSC as shown in Fig. 4.2, (4.9) and (4.10) become

$$G_{S1} : \begin{bmatrix} \Delta f_1 \\ \Delta \dot{P}_{T12} \\ \Delta \dot{P}_{SSSC12} \end{bmatrix} = \begin{bmatrix} -D_1/M_1 a_{S12} & a_{12}/M_1 \\ -2\pi T_{12} & 0 & 0 \\ 0 & 0 & -1/T_{SSSC12} \end{bmatrix} \begin{bmatrix} \Delta f_1 \\ \Delta P_{T12} \\ \Delta P_{SSSC12} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1/T_{SSSC12} \end{bmatrix} \Delta u_{SSSC12}$$
(4.11)

and

$$G_{S2} : \begin{bmatrix} \Delta \dot{P}_{723} \\ \Delta \dot{f}_{3} \\ \Delta \dot{P}_{SSSC23} \end{bmatrix} = \begin{bmatrix} -D_{1}/M_{1} a_{S12} & a_{12}/M_{1} \\ -2\pi T_{12} & 0 & 0 \\ 0 & 0 & -1/T_{SSSC23} \end{bmatrix} \begin{bmatrix} \Delta P_{723} \\ \Delta f_{3} \\ \Delta P_{SSSC23} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1/T_{SSSC23} \end{bmatrix} \Delta u_{SSSC23}$$
 (4.12)

where Δu_{SSSC12} and Δu_{SSSC23} are output signals of frequency stabilizers for SSSC12 and SSSC23, respectively. Equations (4.11) and (4.12) are used to design SSSC12 and SSSC23, respectively.

4.3.4 Structure of Discrete-based Frequency Stabilizer

In this study, the structure of the frequency stabilizer is based on a second-order lead/lag discrete compensator as shown in Fig. 4.3. There are five parameters for each designed frequency stabilizer consisting of a stabilization gain K, time constants T_1, T_2, T_3 , and T_4 .

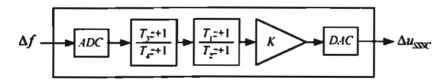


Figure 4.3: Configuration of 2nd order discrete-based frequency stabilizer

For the brief design procedure, first, the control parameters of the continuous-based frequency stabilizer are optimized by a tabu search algorithm. Next, the continuous-based frequency stabilizer is converted to the discrete-based frequency stabilizer by using z-transformation [33]. In this study, the sampling time is set to 0.01 sec.

4.3.5 Formulation of Objective Function

In deriving the objective function, not only the enhancement of system damping, but also the robust stability against system uncertainties are taken into account. Since the main purpose of the designed frequency stabilizer is to improve the system damping following any load disturbances, therefore, the damping ratio (ζ) of the inter-area mode is used as a design specification. Assuming that the eigenvalues corresponding to the mode of oscillation can be determined as $-\sigma \pm j\omega_d$, the damping ratio is given by

$$\zeta_{actual} = \frac{-\sigma}{\sqrt{\sigma^2 + \omega_d^2}} \tag{4.13}$$

The desired damping ratio of the eigenvalues corresponding to the mode of oscillation is specified as $\zeta_{desired}$. Accordingly, the difference between the desired and the actual damping ratios can be defined as

$$\alpha = |\zeta_{desired} - \zeta_{actual}| \tag{4.14}$$

For robust stability, the system uncertainties are modeled as a multiplicative form demonstrated in Fig. 4.4. \tilde{G} is a system and \tilde{K} is a designed controller. Δ_m is a stable multiplicative uncertainty.

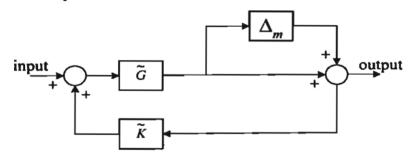


Figure 4.4: Control system with multiplicative uncertainty model

Based on the small-gain theorem, the closed loop system will be robustly stable if

$$\left|\Delta_{m}\right| < \frac{1}{\left|\tilde{G}\tilde{K}\left(1-\tilde{G}\tilde{K}\right)^{-1}\right|} \tag{4.15}$$

where $\tilde{G}\tilde{K}\left(1-\tilde{G}\tilde{K}\right)^{-1}$ is the complementary sensitivity function (T). Note that $|\bullet|$ is the magnitude of the transfer function " \bullet ".

Based on the multiplicative uncertainty model, the robust stability margin can be guaranteed in terms of multiplicative stability margin (MSM) as

$$MSM = 1/||T||,$$
 (4.16)

where $||T||_{x}$ is the ∞ -norm of T. From (4.15) and (4.16), it is clear that by minimizing $||T||_{\infty}$, the MSM increases and the robust stability is ensured. Thus, the normalized robustness index of the objective function is defined as

$$\gamma = ||T||_{\infty}/||T||_{\alpha(miial)} \tag{4.17}$$

where $||T||_{x(mind)}$ is the ∞ -norm of T at the initial of a search process.

Combining (4.14) and (4.17), the control problem can be formulated as the following optimization problem:

Minimize
$$F(K, T_i) = c \cdot \alpha + \gamma$$

Subject to $K_{\min} \leq K \leq K_{\max}$ (4.18)
 $T_{i, \min} \leq T_{i, \max}, \quad i = 1,..., 4$

where $F(K, T_i)$ is the objective function. The minimum and maximum values of the gain K are set to 0.1 and 5, respectively. The minimum and maximum values of the time constants T_i are set to 0.01 and 2, respectively. The constant coefficient "c" is used to weight α -term, so that $c \cdot \alpha$ dominates γ during the parameters optimization. Note that, since γ is normalized to 1 at the initial solution, it is easy to find the value of c so that $c \cdot \alpha$ is greater than 1. Eventually, the search process minimizes both terms until $c \cdot \alpha$ meets the design specification and γ decreases to the possible minimum value. All searched parameters are optimized by a tabu search algorithm.

4.3.6 TSA for Parameter Determination

Tabu Search (TS) is an iterative improvement procedure that can start from any initial feasible solution (searched parameters) and attempted to determine a better solution. As a meta-heuristic, TS is based on a local search technique with the ability to escape from being trapped in local optima. Hereafter, components of TS and TS procedure are discussed.

Encoding and Decoding: The concatenated encoding method is used to encode each parameter into a binary string normalized over its range and also stack each normalized string in series with each other to construct the string individual. The same number of *nb* bits is used for each searched parameter. Figure 4.5 illustrates the example of concatenated encoding scheme.

Figure 4.5: Example of Concatenated Encoding Scheme

On the other hand, a decoding scheme is carried out by converting encoded parameters to their actual values by (4.19) prior to evaluation of objective function.

$$P_{i} = P_{i,\min} + \frac{B_{i} \times [P_{i,\max} - P_{i,\min}]}{2^{n} - 1}$$
 (4.19)

where P_i is the actual value of the *i*-th parameter, $P_{i,max}$ and $P_{i,min}$ are the maximum and minimum value of the *i*-th parameter. B_i is the decimal integer value of binary string of the *i*-th parameter. In this study, 16 bits are used to represent each parameter. The more the number of bits per searched parameter is, the higher the resolution will be.

Trial Solution Generation: To generate a trial solution, one bit of a binary string of an initial solution is flipped at a time. Figure 4.6 conceptually illustrates the process. The maximum number of trial solutions in each iteration is referred to a neighborhood solution space (NS). In this paper, NS is set to 90 % of a total number of bits in a string individual $(\lfloor 0.9 \times nb \times NP \rfloor)$ where NP is a number of searched parameters.

Initial solution: 1001011010 1100011010 1110010011

Trial solution 1: 2001011010 1100011010 1110010011

Trial solution 2: 1101011010 1100011010 1110010011

Trial solution 3: 1011011010 1100011010 1110010011

Figure 4.6: Concept of Trial Solution Generation

Tabu List Restriction: Tabu List (TL) is utilized to keep attributes (bit positions) that created the best solution in the past iterations for iterations so that they can not be used to create new solution candidates. As the iteration proceeds, TL stores a new attribute and releases the oldest one, as shown in Fig. 4.7. Particularly, the size of TL is the only control parameter of TS. The size of TL that provided good solutions usually grows with the size of the problem. In this paper, $|\sqrt{nb \times NP}|$ is used to determine the best size of TL.

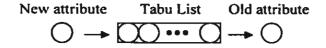


Figure 4.7: Mechanism of tabu list

Aspiration Level Criterion: The aspiration level (AL) criterion allows an attribute included in TL to override its tabu status if it leads to a more attractive solution. In particular, the AL is satisfied if the tabued attribute yields a solution that is better than the best solution reached at that iteration. After the AL is satisfied, updating TL is carried out by moving the tabued attribute back to the first position of the TL.

Termination Criteria: This criterion is set to allow the search process to stop and return the best solution found. The search process will terminate if the maximum allowable number of iterations is reached.

TS Procedure: firstly, the initial feasible solution is generated arbitrarily. A trial solution is searched if either it is not tabued or, in case of being tabued it passes the AL test. The best solution is always updated during the search process until the termination criterion is satisfied. The following notations are used for the TS procedure.

TL: the tabu list,

NS: the neighborhood solution space,

F(X): the objective function of solution X,

 X_a^k : the initial feasible solution at iteration k,

 X_m^k : a trial solution m at iteration k,

 X_{cb}^{k} : the current best trial solution at iteration k,

 X_b^k : the best solution reached at iteration k,

 k_{max} : the maximum allowable number of iterations.

The TS procedure can be described as follows.

- I. Read constraints of searched parameters, the initial feasible solution X_o^k , and the design specification.
- 2. Specify the size of TL, k_{max} , and size of NS.
- 3. Initialize iteration counter k and termination criteria tc to zero, and empty TL.
- 4. Initialized AL by setting $X_b^k = X_o^k$.
- 5. Execute TS procedure:

- 5.1 Initialize the trial counter m to zero.
- 5.2 Generate a trial solution X_m^k from X_o^k .
- 5.3 If X_m^k is not feasible, go to 5.9.
- 5.4 If X_m^k is the first feasible solution, set $X_{ch}^k = X_m^k$.
- 5.5 Perform Tabu test. If X_m^k is tabued, then go to 5.8.
- 5.6 If $F(X_m^k) < F(X_{cb}^k)$, set $X_{cb}^k = X_m^k$. Otherwise, go to 5.9.
- 5.7 If $F(X_m^k) < F(X_b^k)$, then update AL by setting $X_b^k = X_m^k$. Go to 5.9.
- 5.8 Perform AL test. If $F(X_m^k) < F(X_b^k)$, set $X_{cb}^k = X_m^k$, and update AL by setting $X_b^k = X_m^k$.
 - 5.9 If m is less than NS, m = m+1 and go to 5.2.
 - 5.10 If there is no feasible solution, set $X_o^{k-1} = X_b^k$. Otherwise, set $X_o^{k+1} = X_{cb}^k$, and update TL.
- 6. If k = 0, go to 9.
- 7. Perform the convergence checking.

If
$$X_b^k = X_b^{k-1}$$
, $tc = tc+1$. Otherwise, $tc = 0$.

- 8. If tc = size of TL, set tc = 0 and go to 10.
- 9. If $k < k_{\text{max}}$, then k = k+1, and go to 5.
- 10. TS is terminated and X_b^k is the best solution found.

4.4 Experimental Results

In the design specification, the desired damping ratio ($\zeta_{desired}$) is set to 0.25. Also, the coefficient c is appropriately set to 5. For TS, the size of TL is set to 8 for the best solution. The area capacity ratio between areas 1, 2 and 3 is 5:10:2. System parameters are given in Table 4.1.